Development of the preventive maintenance system for belt conveyors reducers

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Abstract. Heavy operating conditions of mining machines as well as the high level of dynamic loads lead to reduction of their service life. The quantitative estimation of the machine reliability by one of the feature service life - has become widely distributed in all the branches of engineering. Technical diagnosis is one of the important methods of improving the reliability in operating conditions. The diagnostics subsystem should provide for: non-destructive inspection of a technical condition of objects, the definition of sudden and parametric failures of mining machines and their systems, the detection of gradual failures by predicting changes in the monitored parameters, a continuous and periodic technical inspection. The obtained results given in this article prove the possibility of creating a group of common diagnostic criteria suitable for assessing the technical state of reducers of mining machines and equipment, but also being a prerequisite for the effective short-term prediction of the parameters under study when developing adaptive mathematical models.

1 Introduction

All the parts of mining machines and equipment can be subdivided into the following groups:

1. The first group includes dipper teeth and cutting lips; bases and canopies of powered supports; augers; bulldozer bars, blades; crawler shoes, idler rollers, pins, bushings, sprockets of crawler undercarriage and other parts with the service life duration depending on abrasive wear;

2. The second group includes slotted and threaded parts, gear couplings, mounting seats for rolling bearings of shafts, machine tools, surfaces of gearing, etc. with the service life duration determined by mechanical wear of their surfaces;

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3. The third group includes parts of internal combustion engines of dump trucks, bulldozers, scrapers, diesel locomotives, thermal rock fracturing and jet drilling rigs, etc. with their service life duration limited by molecular – chemical or corrosive mechanical wear;

4. The fourth group includes rolling and plain bearings, shock absorbers, springs, connector rods, connecting rod bolts, etc. with their surface life duration depending on metal fatigue strength.

2 Formulation of the Problem

At present, the coal mines operate a significant number of belt conveyors [1, 2], and the performance indicators of the entire coal industry of Kuzbass depend on their operating condition. In the nearest future, it is expected that power consumption and technical equipping of belt conveyors increase, as well as the performance and length of rock mass transportation, variable frequency drives are widely introduced.

The increasing volumes of coal production by integrated mechanized coal faces, together with an increase in production safety require the creation of reliable transport systems. High efficiency and trouble-free performance along with reduction of energy consumption present the main challenge faced by manufacturers of production lines of mine belt conveyors. Another no less important goal is reduction of the cost of their maintenance and repairs [3, 4]. To ensure reliable operation of a belt conveyor for as long as possible, it is required to find the causes of failure of the various constituent elements [5-7]. The analysis of downtime caused by failure of a belt conveyor reducer shows that it varies from 7.4% to 18.2% and accounts on average for 12%, with the average recovery time is from 24 to 48 hours. It should be noted that the most common failure is belt rupture (up to 50%), while the average time to fix this failure is from 1.5 to 2 hours. Hence, identification of the actual technical condition of mine belt conveyor reducers is really quite an urgent objective.

3 Results of Study

The vibration control method [8-10] proved to be useful in checking the technical condition of mechanical equipment. The vibration diagnostics is used to monitor the current state of equipment, to identify possible defects, to assess the remaining service and to define repair time and scope. The analysis of domestic and foreign experience of monitoring the technical condition of systems with rotational motion of power assemblies shows that for the detection of potential failures the most effective (77%) is control of the machinery condition by vibration parameters [11], and with the involvement of other methods of the functional diagnostics – like spectral oil analysis [12] and thermal control, the accuracy of recognition of the cause of the defect increases to 95%.

The functional diagnostics, in terms of safe operation of mining machinery, should play a key role in the field of research and development, production and quality control of the process equipment. And while in the coal mining and ore mining industries the forms of technical service of technological equipment based on its actual condition [1, 8] are becoming more common, in the manufacture of such equipment new forms of quality control are not yet implemented in spite of the introduction of quality standards GOST ISO 9000-2011. Defects that arise in the gearbox manufacturing process can be divided into errors in the manufacture of gearbox elements and errors in gearbox assembly.

A full analysis of the technical condition of the gearbox after assembly and trial running on the test bench will allow us to identify and localize manufacturing defects, but also to eliminate possible supply of poor quality products to the consumer. In addition, the findings could form the basis for development of automated quality control system.

The analysis of vibration control methods allows us to conclude that it is appropriate to apply the method of support masks for automation of control of products manufactured for the coal industry. This method is based on the fact that the defects formed as a result of assembly operations generate a vibration in certain frequency bands with a certain ratio of the values of controlled parameters. The method of support masks makes it possible to set the width of the frequency band, its position and the values of the evaluation criteria, which are compared with the current values in random order. Based on the analysis of the controlled parameter changes in the frequency band (the number of bands may vary from 6 to 30), the condition of the equipment is evaluated and forecasted [4].

Frequency ranges of the spectrum mask (band width) are usually taken based on the following conditions:

1. "High-energy" components of the spectrum accompanying the misbalance or misalignment – $(0,5...1,5) \times f_r$ and $(1,5...2,5) \times f_r$;

2. "Low-energy" components of vibrations accompanying the defects of rolling bearings – $(7,5...15,5) \times f_r$;

3. $(2,5...10,5) \times f_r$ total failure of the system stiffness;

- 4. The first medium frequency band $(3...15) \times f_r$;
- 5. The second medium frequency band $(15...40) \times f_r$;
- 6. The first high frequency band 40×fr...20 kHz;
- 7. $(0,1...0,9) \times f_r$ for detection of oil wedge defects of slide bearings;
- 8. $(n\pm 1) \times f_r$ for detection of damages in the components of couplings.

The use of modern automation technologies for control of equipment technical condition enables an individual approach to each manufactured mechanism in the evaluation of its technical condition and sets the thresholds of the initial operable and limit state. As an example, Figures 1 and 2 show the spectrum of the vibration signal and its averaged spectral mask in 1 checkpoint of gearbox RKC-400 manufactured by JSC "Anzheromash" (Fig. 3). The measurements were made by analyzers "Corvet", and the signal was processed on Safe Plant software platform developed by NGO "Diatekh".

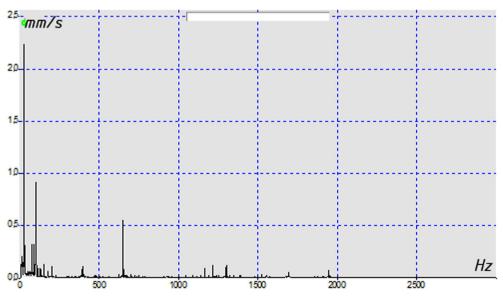


Fig. 1. Vibration signal spectrum

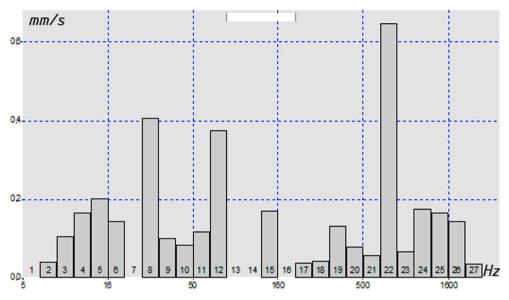


Fig. 2. Vibration signal spectrum mask

In the presented example, the frequency range (2; 3000 Hz) is divided into 27 bands, each of which is normalized by the RMS value of vibration speed V determined for both forward and reverse rotation of the output shaft.

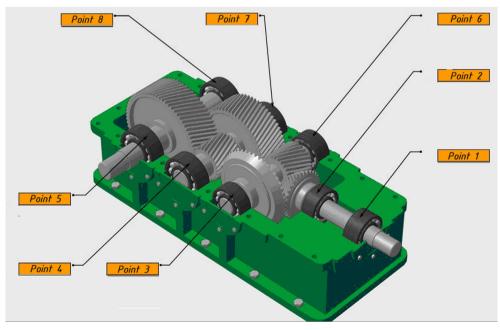


Fig. 3. Vibration measurement points on gearbox RKC-400

The most typical defects inherent to reducers of mining machines occur at frequencies shown in Tables 1 and 2.

Frequency	Type of manufacturing defect	Type of assembly defect	Type of wear defect
f_r	Misbalance		
$ \begin{array}{c} k \times f_{r1 \ \text{and}} k \times f_{r2} \\ (k = 1, 2, \text{ less often 3 and 4}), \\ m \times f_z \pm n \times f_r \\ (m, n = 1, 2) \end{array} $	Variable gear pitch error	Violation of alignment (misalignment of shafts)	
$k \times f_r$ k = 1, 220 and higher		Increased gear backlash	
f_z	Permanent gear pitch error		
$k \times f_{r}, k \times f_{r}$ growth of noise component $m \times f_{m} \pm n \times f_{r}$ (m, n=1,2)			Abrasive wear
$ \begin{array}{l} k \times f_r, \ m \times f_z \pm n \times f_r, \\ m \times f_m \pm n \times f_r \\ \text{(amplitude fluctuation,} \\ n = 0, 1, 2) \end{array} $			Teeth chipping
$ \begin{array}{l} k \times f_r, \\ m \times f_z \pm n \times f_r, \\ m \times f_m \pm n \times f_r, \\ (\text{amplitude fluctuation,} \\ n = 0, 1, 2), \\ \text{growth of noise component} \end{array} $			Cracks and (or)broken teeth

Table 1. Frequencies typical for gearing defects in in-line reducers

Table 2. Defects in the gear part of the planetary reducer and their key diagnostic parameters

Type of defect	Diagnostic signs
Sun pinion shaking	$f_o, nf^* \pm f_o, kf_z \pm f^*$
Sun pinion misalignment	$2f_o, 2nf^* \pm 2f_o, kf_z \pm 2f^*$
Sun pinion teeth defect	$knf^* \pm k_1 f_o, \ kf_z \pm k_l f^*$
Satellite gear misalignment	$4f_g \pm k_1 f_v, \ kf_z \pm 2f_g$
Satellite gear teeth defect	$2kf_g \pm k_1 f_{\nu}, \ kf_z \pm k_l f_g$
Crown misalignment	$2nf_{v}, kf_{z} \pm 2nf_{v}$
Crown teeth defect	$knf_{v}, kf_{z} \pm k_{1}nf_{v}$
Engagement defect	kf _z
Shaking of final drive carrier	$kf_{\nu}, f_o \pm f_{\nu}, kf_z \pm k_{\rm l}f_{\nu}$
Satellite gear bearing defect	$kf_{v}, f_{o} \pm f_{v}, kf_{z} \pm f_{g}/2$
Defect of sun pinion bearing	kf_r + growth of RMS MF, impact pulses appear at MF
Bearing lubrication defect	Impact pulses appear at HF, growth of RMS at HF

In industrial conditions (mine Taldinskaya-Zapadnaya of OJSC "SUEK-Kuzbass") an integrated method of the technical state estimation has been tested on the drive units of mine belt conveyor 3LL1600 (conveying length L = 850 m, technical performance Q = 3500 m / h, the belt speed v = 0 -4 m / s) by the parameters of the lubricating oil, vibration and thermal control.

4 Conclusion

The proposed approach to the standardization of mechanical vibration parameters can be used in practice at development of the enterprise standard for normalization of vibration of manufactured products for inclusion in the product data sheet.

The development of a large number of spectrum masks for a wide standard line of mining equipment is one of the prerequisites for release by mining machinery plants of quality products and the transition to new forms of mining machines maintenance and repair.

The results of evaluation of the technical condition of the 3LL1600 conveyor drives using the integrated method based on monitoring of parameters of vibro-acoustic signals, emission-spectral analysis of the composition of working oil and thermal visual control of support units of rolling bearings make it possible to track changes in the technical condition of the reducer elements depending on its load and speed.

The proposed approach will not only increase the accuracy of the assessment of the technical condition of mining equipment gear units but will also help to organize the work on creation of normative and methodological base for building predictive models of changes in the technical condition on the basis of a significant amount of accumulated statistical information on the development of those or other defects of belt conveyor reducers.

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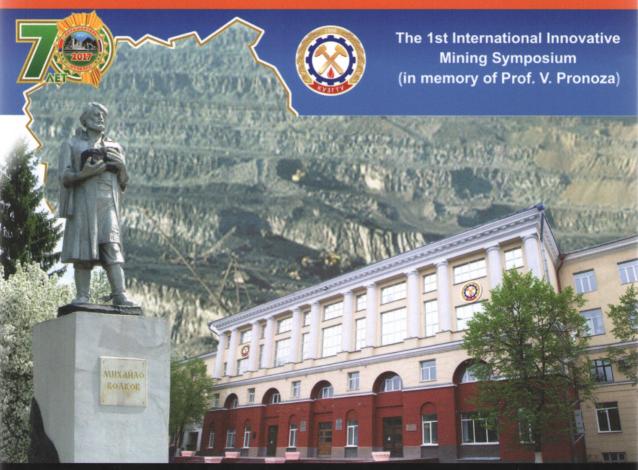
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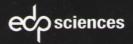
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Contents

00001 Preface: the role of T.F. Gorbachev State Technical University as the Flagship of Kemerovo Region Innovative Development *A. Krechetov*

Perspective Mining Technologies

- 01001 The Length Of Bearing Pressure Zone For The Flat Seams Extraction In A Linear Deformation Rock
 V. Gogolin, Y. Lesin and A. Djagileva
- 01002 The influence of advancing speed of powered mining stope with single face on earth's surface displacing in Kuzbass A. Renev, S. Svirko, A. Bykadorov and V. Fedorin
- 01003 Energy Consumption in the Process of Excavator-Automobile Complexes Distribution at Kuzbass Open Pit Mines I. Panachev, B. Gerike, I. Kuznetsov and A. Shirokolobova
- 01004 Slope Stability Assessment and Underground Mine Design Analysis of Achibo-Sombo Underground Conventional Coal Mine, Southwest Ethiopia *M. Haftu, B. Konka, K. Woldeargay and A. Abay*
- 01005 Geomechanics of rock array for chamber system of coal deposits development on the example of finalizing by KGRP complex
 A. Bykadorov, D. Degtyarev, S. Smirnov and O. Pechenegov
- 01006 Justification of the Optimal Granulometric Composition of Crushed Rocks for Open-Pit Mine Road Surfacing V. Shalamanov, V. Pershin, S. Shabaev and D. Boiko
- 01007 Optimization of transportless technological schemes for coal seams quarrying T. Gvozdkova, E. Plotnikov and E. Usova
- 01008 The features of three- and four-tier internal dumps capacity calculation with the additional capacity preparation in the dump tiers T. Gvozdkova, E. Kuznetsov, A. Rudakova and S. Markov
- 01009 The Relationship Between the Manifestations of Rock Pressure and the Relative Deformation of Surrounding Rocks
 S. Kostyuk, N. Bedarev, O. Lyubimov and A. Shaikhislamov
- 01010 Computer Simulation of Electroosmotic Soils Treatment M. Gucal and A. Pokatilov
- 01011 Innovative numerical modelling of technogenic rock arrays structure S. Markov, M. Tyulenev, O. Litvin and E. Tyuleneva
- 01012 Definition of the form of coal spontaneous combustion source as the inverse problem of geoelectrics

D. Sirota, V. Ivanov and V. Khyamyalyaynen

- 01013 Numerical Simulation of Primary Roof Collapse in Production Workings I. Ermakova and V. Klimov
- 01014 The Study of Processes of Electrochemical Treatment of Soils at the Pilot Test Site S. Prostov, E. Shabanov and A. Shadrin
- 01015 The study of stress-strain state of stabilized layered soil foundations M. Sokolov, S. Prostov and V. Zykov
- 01016 Justification of parameters and technology of retaining prism filling to eliminate landslide
 V. Balakhnin, O. Veretennikova, R. Pobegaylo and E. Mezina
- 01017 Use of Deep Peat-Processing Products for Hydrophobic Modification of Gypsum Binder
 - O. Misnikov and V. Ivanov
- 01018 Software for Automated Production Line of Peat Briquettes V. Lebedev and O. Puhova
- 01019 Assessment process of concept for mining and its impact on the region M. Cehlår, J. Janočko, Z. Šimková and T. Pavlik
- 01020 Experimental Study of Methane Hydrates in Coal V. Smirnov, V. Dyrdin, T. Kim, A. Manakov and A. Khoreshok
- 01021 Expert evaluation of innovation projects of mining enterprises on the basis of methods of system analysis and fuzzy logics
 A. Pimonov, E. Raevskaya and T. Sarapulova
- 01022 Gas hydrates in coal seams and their impact on gas-dynamic processes in underground mining *T. Kim, V. Dyrdin, V. Smirnov and V. Nesterov*

Environment Saving Development of Mining

- 02001 Environmental and Economic Efficiency of Comprehensive Technology of Sulfur Oxides, Nitrogen Oxides and Mercury Removal from Flue Gases S. Grigashkina, T. Galanina, V. Mikhailov, T. Koroleva and E. Trush
- 02002 Utilization prospects for coal mine methane (CMM) in Kuzbass O. Tailakov, D. Zastrelov, V. Tailakov, M. Makeev and P. Soot
- 02003 The Experience of Using Innovative Artificial Filter Arrays on South Kuzbass Open Pit: Case Study M. Tuulanay, Y. Lacin, F. Tuulanaya and F. Murko
 - M. Tyulenev, Y. Lesin, E. Tyuleneva and E. Murko
- 02004 The results of air treatment process modeling at the location of the air curtain in the air suppliers and ventilation shafts

A. Nikolaev, N. Alymenko, A. Kamenskih and V. Nikolaev

02005 Mitigating Against Conflicts in the Kenyan Mining Cycle: Identification of Gaps in the Participation and Recourse for Rights Holders (Civil Society & Community) S. Anyona and B. Rop

Innovative Mining Equipment

- 03001 Modeling of Power Consumption of the Mining Equipment Using "The Probabilistic Automata Method"
 - A. Zakharova, V. Kashirskikh, I. Lobur, N. Shauleva and V. Borovtsov
- 03002 Application of Machine Learning for Dragline Failure Prediction A. Taghizadeh and N. Demirel
- 03003 Kinematic Parameters Of Rotary Transmission With Hydraulic Cylinders M. Blaschuk, A. Dronov, A. Koperchuk, R. Chernukhin and V. Litvinenko
- 03004 The results of cutting disks testing for rock destruction A. Khoreshok, L. Kantovich, V. Kuznetsov, E. Preis and D. Kuziev
- 03005 Grinding efficiency improvement of hydraulic cylinders parts for mining equipment A. Korotkov, V. Korotkov, L. Mametyev, L. Korotkova and T. Terjaeva
- 03006 The Smart Grid using in the Kuzbass open-pit coalmine I. Semykina, A. Evstratov and G. Lebedev
- 03007 Multifunctional Testing Rig for Machinery Safety Equipment S. Vöth, J. Tschersich and Tim Schwartz
- 03008 Development of the preventive maintenance system for belt conveyors reducers B. Gerike, I. Panachev and E. Kuzin
- **03009** Determination of Load Performance of Two-Bar Girder Lining Y. Glazkov, A. Kazantsev, D. Nesteruk, V. Aksenov and A. Efremenkov
- **03010** Ways of increasing excavator fleet productivity in Russian coal open pits (Kuzbass case study)

M. Drygin, N. Kurychkin and A. Bakanov

03011 Strategy of Russian Coal Mining Enterprises' Excavator Park Technical State Correction

M. Drygin, N. Kurychkin and A. Bakanov

- 03012 The innovative development of machine building as a driver of import substitution S. Zhironkin, M. Gasanov, G. Barysheva, K. Kolotov and O. Zhironkina
- 03013 "Smart Service" as an innovative system of service for mining companies in Kuzbass
 - L. Samorodova, L. Shut'ko, Y. Yakunina and O. Lyubimov
- 03014 Special Modes of AC Drives Operation in the Mining Industry

L. Payuk, N. Voronina, O. Galtseva, D. Zhang and A. Rogachev

- 03015 Impact of the number of blades of the geokhod cutting body on cutting forces V. Aksenov, V. Sadovets, E. Rezanova and D. Pashkov
- 03016 Comparison of technological efficiency of gravitational devices for preparation of large diluted coal

V. Udovitsky, V. Kandinsky and A. Begunov

03017 The intelligent mechatronic system for open pit mining to increase the operation life of equipment

I. Semykina, V. Zavyalov and V. Kashirskikh

Economic and Social Development of Mining Regions

- 04001 The principles of municipal industrial clusters' establishment on the territory of advancing social-and-economic development of mono-town
 - O. Ivanova, G. Antonov and S. Bereznev
- 04002 The innovative strategy of social and economic development of mining region S. Bereznev, O. Zonova and E. Lubkova
- 04003 The analysis of strategies for the mining regions' development in Russia as a condition of effective management of economy N. Zaruba, N. Egorova and P. Kosinskij
- 04004 Increasing the efficiency of coal mining based on the concept of Shewhart-Deming variability management

V. Mikhalchenko and Y. Rubanik

- 04005 Evaluation of consumer satisfaction with the quality of training of young professionals by the universities for enterprises of coal-mining complex *V. Mikhalchenko and I. Seredkina*
- 04006 Innovative development of the economy as the most important factor in ensuring the financial security of the mining region

N. Kudrevatykh, T. Snegireva and A. Tselischeva

- 04007 Theoretical Foundations of the New Industrialization of the Mining Region under Globalization
 - L. Kusurgasheva, O. Nedospasova and E. Zhernov
- 04008 Problems and Prospects of Sustainable Development of Mining Regions I. Levitskaya, N. Pastukhova and O. Dubrovskaya
- 04009 Innovations as borders of stages of coal industry historical development E. Sigareva, S. Popov, S. Baturin, N. Sidorova and M. Borisova
- 04010 Using innovative interactive technologies for forming linguistic competence in global mining education
 - G. Chistyakova, E. Bondareva, K. Demidenko, E. Podgornaya and O. Kadnikova

04011 Technological convergence and innovative development of natural resource economy

F. Agafonov, A. Genin, O. Kalinina, O. Brel and O. Zhironkina

04012 Structural problems of mining region innovative development (Kuzbass, Western Siberia)

E. Dotsenko and N. Ezdina

04013 Neo-industrialization of Kuzbass economy in innovative development of coal industry and machinery

A. Balabanova, V. Balabanov, E. Dotsenko and N. Ezdina

04014 The problems of correlation the life quality and interpersonal dialogue in legal practice of mining regions

V. Zolotukhin, E. Stepantsova, M. Kozyreva, A. Tarasenko and A. Stepantsov

04015 Demographic and migration policy in the mining region and its impact on the ecological consciousness of the population

V. Zolotukhin, A. Bel'kov, E. Stepantsova, M. Kozyreva and A. Tarasenko

04016 Cross-cultural analysis of the verbal conflict behavior of the graduate mining engineers

I. Pevneva, O. Gavrishina, A. Rolgayzer, M. Agienko and A. Myaskov

- 04017 The impact of human factor on labor productivity at the mining enterprises G. Pinigina, I. Kondrina, S. Smagina, V. Tatsienko and A. Meshkov
- 04018 Enhancing the Role of Educational Services of Higher Education System in the Competitive Specialists Training for Industry O. Kuznetsova, S. Kuznetsova, E. Yumaev, V. Kuznetsov and I. Plotnikova
- 04019 Formation and Development of the Training System for Innovative Development of Regional Industry

O. Kuznetsova, S. Kuznetsova, E. Yumaev, V. Kuznetsov and O. Galtseva

04020 Evaluating the Effectiveness of Internal Corporate Controls in Coal Mines Illustrated By the Example of JSC "SUEK-Kuzbass"

E. Kucherova, T. Ponkratova, T. Tyuleneva and N. Cherepanova

04021 The communication aspect of specialists' professional competence L. Znikina, N. Mamontova and P. Strelnikov

volume 15 - 2017

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